

Control of SCOLE

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THIRD ANNUAL SCOLE WORKSHOP

CONTROL OF SCOLE

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MODAL CONTROL

The object is to control the SCOLE using a relatively low order model.

Discretized model: $\ddot{M}\underline{q}(t) + K\underline{q}(t) = \underline{F}(t) + \underline{v}(t)$

$\underline{q}(t)$ = relatively high-dimensional configuration vector

$\underline{v}(t)$ = actuator noise vector

Drastic truncation of the model is proposed by means of a modal expansion.

Open-loop eigenvalue problem: $K\underline{u}_i = \omega_i^2 M\underline{u}_i, \quad i = 1, 2, \dots, n$

Eigenvalue orthonormality: $\underline{u}_j^T M \underline{u}_i = \delta_{ij}, \quad \underline{u}_j^T K \underline{u}_i = \omega_i^2 \delta_{ij}$

Modal truncation: $\underline{q}(t) = \sum_{i=1}^c \underline{u}_i \eta_i(t) = U_c \underline{\eta}(t)$

$U_c = [\underline{u}_1 \ \underline{u}_2 \ \dots \ \underline{u}_c]$ = truncated modal matrix

$\underline{\eta}(t)$ = c-dimensional modal vector

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MODAL CONTROL (CONT'D)

Truncated modal equations: $\ddot{\eta}_i(t) + \omega_i^2 \eta_i(t) = f_i(t) + v_i(t), \quad i = 1, 2, \dots, c$

$$f_i(t) = \tilde{u}_i^T F(t) = \text{modal control}$$

$$v_i(t) = \tilde{u}_i^T v(t) = \text{modal actuator noise}$$

Modal state equations: $\dot{\tilde{x}}_i(t) = A_i \tilde{x}_i(t) + B_i [f_i(t) + v_i(t)], \quad i = 1, 2, \dots, c$

$$A_i = \begin{bmatrix} 0 & 1 \\ -\omega_i^2 & 0 \end{bmatrix}, \quad B_i = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

Actual output vector: $\tilde{y}(t) = C\tilde{x}(t) + \tilde{w}(t)$

$C = s \times 2c$ matrix with c elements of a given row obtained from U_c
and the balance equal to zero

$\tilde{x}(t) = \text{overall modal state}$

$\tilde{w}(t) = \text{measurement (sensor) noise vector}$

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MODAL CONTROL (CONT'D)

Modal Kalman filter: $\dot{\tilde{x}}(t) = \hat{A}\tilde{x}(t) + B\tilde{f}(t) + K(t)[y(t) - \hat{C}\tilde{x}(t)]$

A = block-diag A_i , B = block-diag B_i

K = estimator gain matrix

To determine the matrix K , it is necessary to solve first a 2×2 matrix Riccati equation for given actuator and sensor noise intensities.

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INDEPENDENT MODAL-SPACE CONTROL (IMSC)

Linear (proportional and rate feedback) control:

$$\dot{f}_i = -h_i \dot{\eta}_i - g_i \eta_i$$

h_i, g_i = modal gains

Nonlinear control (on-off):

$$\dot{f}_i = -k_i, \eta_i \geq d_i; \quad 0, |\dot{\eta}_i| < d_i; \quad k_i, \eta_i \leq -d_i$$

$2d_i$ = width of the deadband region

k_i = magnitude of the modal control force

INDEPENDENT MODAL-SPACE CONTROL (IMSC) (CONT'D)

Synthesis of actual controls: let the number of controlled modes coincide with the number of actuators.

Because $\tilde{F}(t)$ is of smaller dimension than $\tilde{q}(t)$, let

$$M\ddot{\tilde{q}}(t) + K\tilde{q}(t) = P[\tilde{F}(t) + \tilde{v}(t)], \quad P = n \times c$$

$$\tilde{f}(t) = U_C^T P \tilde{F}(t) + \tilde{F}(t) = (U_C^T P)^{-1} \tilde{f}(t)$$

∴ The components of $\tilde{F}(t)$ are linear combinations of the components of $\tilde{f}(t)$. When the modal control is nonlinear, the components of $\tilde{F}(t)$ are quantized and have the appearance of staircase functions

